

# Rules and Regulations

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## DEPARTMENT OF ENERGY

### 10 CFR Part 430

[Docket Number EERE-2014-BT-TP-0014]

RIN 1904-AD22

#### Energy Conservation Program: Test Procedures for Portable Air Conditioners; Correction

**AGENCY:** Office of Energy Efficiency and Renewable Energy, Department of Energy.

**ACTION:** Correcting amendments.

**SUMMARY:** The U.S. Department of Energy (DOE) published a final rule in the *Federal Register* on June 1, 2016, establishing test procedures for portable air conditioners. This correction addresses typographical errors in that final rule that were included in Title 10 of the Code of Federal Regulations (CFR) part 430, subpart B, appendix CC. Neither the errors nor the corrections in this document affect the substance of the rulemaking or any of the conclusions reached in support of the final rule.

**DATES:** This correction is effective October 14, 2016.

**FOR FURTHER INFORMATION CONTACT:** Mr. Bryan Berringer, U.S. Department of Energy, Office of Energy Efficiency and Renewable Energy, Building Technologies Office, EE-5B, 1000 Independence Avenue SW., Washington, DC 20585-0121. Telephone: (202) 586-0371. Email: [Bryan.Berringer@ee.doe.gov](mailto:Bryan.Berringer@ee.doe.gov).

Ms. Sarah Butler, U.S. Department of Energy, Office of the General Counsel, Mailstop GC-33, 1000 Independence Ave. SW., Washington, DC 20585-0121. Telephone: (202) 586-1777. Email: [Sarah.Butler@hq.doe.gov](mailto:Sarah.Butler@hq.doe.gov).

**SUPPLEMENTARY INFORMATION:** On June 1, 2016, DOE published a final rule (the "June 2016 final rule") to establish test procedures for portable air conditioners.

81 FR 35241. DOE has since found that the June 2016 final rule contained minor typographical errors in Title 10 of the Code of Federal Regulations (CFR) part 430, subpart B, appendix CC. This final rule correction revises appendix CC to subpart B of 10 CFR part 430, to correct these typographical errors. Specifically, in section 4.1.1, DOE is correcting the following errors: An incorrect subscript for the variable  $T_{duct\_SD\_j}$  in the  $Q_{duct\_SD}$  equation and missing subscripts "j" on the  $T_{duct}$  variables in the equations for  $Q_{duct\_95}$  and  $Q_{duct\_83}$ . In section 4.1.2, DOE is correcting the following errors: A missing equals sign and parenthesis; incorrect subscripts for the variable  $C_{p\_da}$  and the infiltration air variables in the  $Q_{s\_95}$  equation; incorrect subscripts in the infiltration air variables in the  $Q_{s\_83}$  equation; missing equals signs in the  $Q_{l\_95}$  and  $Q_{l\_83}$  equations; and missing "Q" variables and incorrect subscripts for the  $Q_{l\_95}$  and  $Q_{l\_83}$  variables in the  $Q_{infiltration\_95}$  and  $Q_{infiltration\_83}$  equations.

DOE also found that the summation symbols in the two dual-duct  $Q_{duct}$  equations in section 4.1.1 were not properly represented in the Electronic Code of Federal Regulations (eCFR).

Neither the errors nor the corrections in this document affect the substance of the rulemaking or any of the conclusions reached in support of the final rule. Accordingly, DOE finds that there is good cause under 5 U.S.C. 553(b)(B) to not issue a separate notice to solicit public comment on the corrections contained in this final rule as doing so would be impractical, unnecessary, and contrary to the public interest. For the same reasons and pursuant to 5 U.S.C. 553(d), DOE finds good cause to waive the 30-day delay in effective date.

#### Procedural Issues and Regulatory Review

DOE has concluded that the determinations made pursuant to the various procedural requirements to the June 2016 final rule that originally codified DOE's test procedures for portable air conditioners remain unchanged for this final rule technical correction. 81 FR 35241. The amendments from that final rule became effective July 1, 2016. *Id.*

#### List of Subjects in 10 CFR Part 430

Administrative practice and procedure, Confidential business information, Energy conservation,

Household appliances, Imports, Intergovernmental relations, Reporting and recordkeeping requirements, and Small businesses.

Issued in Washington, DC, on October 7, 2016.

**Kathleen Hogan,**

*Deputy Assistant Secretary for Energy Efficiency, Energy Efficiency and Renewable Energy.*

For the reasons set forth in the preamble, DOE amends part 430 of title 10, Code of Federal Regulations by making the following correcting amendments:

#### PART 430—ENERGY CONSERVATION PROGRAM FOR CONSUMER PRODUCTS

■ 1. The authority citation for part 430 continues to read as follows:

**Authority:** 42 U.S.C. 6291–6309; 28 U.S.C. 2461 note.

■ 2. Appendix CC to subpart B of part 430 is amended by revising sections 4.1.1 and 4.1.2 to read as follows:

#### Appendix CC to Subpart B of Part 430—Uniform Test Method for Measuring the Energy Consumption of Portable Air Conditioners

\* \* \* \* \*

4. \* \* \*

4.1.1. *Duct Heat Transfer.* Measure the surface temperature of the condenser exhaust duct and condenser inlet duct, where applicable, throughout the cooling mode test. Calculate the average temperature at each individual location, and then calculate the average surface temperature of each duct by averaging the four average temperature measurements taken on that duct. Calculate the surface area ( $A_{duct\_j}$ ) of each duct according to:

$$A_{duct\_j} = \pi \times d_j \times L_j$$

Where:

$d_j$  = the outer diameter of duct "j", including any manufacturer-supplied insulation.

$L_j$  = the extended length of duct "j" while under test.

j represents the condenser exhaust duct and, for dual-duct units, the condenser exhaust duct and the condenser inlet duct.

Calculate the total heat transferred from the surface of the duct(s) to the indoor conditioned space while operating in cooling mode for the outdoor test conditions in Table 1 of this appendix, as follows. For single-duct portable air conditioners:

$$Q_{duct\_SD} = h \times A_{duct\_j} \times (T_{duct\_SD\_j} - T_{ei})$$

For dual-duct portable air conditioners:

$$Q_{\text{duct}_{95}} = \sum_j \{h \times A_{\text{duct}_{j}} \times (T_{\text{duct}_{95}_{j}} - T_{ei})\}$$

$$Q_{\text{duct}_{83}} = \sum_j \{h \times A_{\text{duct}_{j}} \times (T_{\text{duct}_{83}_{j}} - T_{ei})\}$$

Where:

$Q_{\text{duct}_{SD}}$  = for single-duct portable air conditioners, the total heat transferred from the duct to the indoor conditioned space in cooling mode when tested according to the test conditions in Table 1 of this appendix, in Btu/h.

$Q_{\text{duct}_{95}}$  and  $Q_{\text{duct}_{83}}$  = for dual-duct portable air conditioners, the total heat transferred from the ducts to the indoor conditioned space in cooling mode, in Btu/h, when tested according to the 95 °F dry-bulb and 83 °F dry-bulb outdoor test conditions in Table 1 of this appendix, respectively.

$h$  = convection coefficient, 3 Btu/h per square foot per °F.

$A_{\text{duct}_{j}}$  = surface area of duct "j", in square feet.

$T_{\text{duct}_{SD}_{j}}$  = average surface temperature for the condenser exhaust duct of single-duct portable air conditioners, as measured during testing according to the test condition in Table 1 of this appendix, in °F.

$T_{\text{duct}_{95}_{j}}$  and  $T_{\text{duct}_{83}_{j}}$  = average surface temperature for duct "j" of dual-duct portable air conditioners, as measured during testing according to the two outdoor test conditions in Table 1 of this appendix, in °F.

$j$  represents the condenser exhaust duct and, for dual-duct units, the condenser exhaust duct and the condenser inlet duct.

$T_{ei}$  = average evaporator inlet air dry-bulb temperature, in °F.

**4.1.2 Infiltration Air Heat Transfer.** Measure the heat contribution from infiltration air for single-duct portable air conditioners and dual-duct portable air conditioners that draw at least part of the condenser air from the conditioned space. Calculate the heat contribution from infiltration air for single-duct and dual-duct portable air conditioners for both cooling mode outdoor test conditions, as described in this section. Calculate the dry air mass flow rate of infiltration air according to the following equations:

$$\dot{m}_{SD} = \frac{V_{CO_{SD}} \times \rho_{CO_{SD}}}{(1 + \omega_{CO_{SD}})}$$

For dual-duct portable air conditioners:

$$\dot{m}_{95} = \left[ \frac{V_{CO_{95}} \times \rho_{CO_{95}}}{(1 + \omega_{CO_{95}})} \right] - \left[ \frac{V_{CI_{95}} \times \rho_{CI_{95}}}{(1 + \omega_{CI_{95}})} \right]$$

$$\dot{m}_{83} = \left[ \frac{V_{CO_{83}} \times \rho_{CO_{83}}}{(1 + \omega_{CO_{83}})} \right] - \left[ \frac{V_{CI_{83}} \times \rho_{CI_{83}}}{(1 + \omega_{CI_{83}})} \right]$$

Where:

$\dot{m}_{SD}$  = dry air mass flow rate of infiltration air for single-duct portable air conditioners, in pounds per minute (lb/m).

$\dot{m}_{95}$  and  $\dot{m}_{83}$  = dry air mass flow rate of infiltration air for dual-duct portable air conditioners, as calculated based on testing according to the test conditions in Table 1 of this appendix, in lb/m.

$V_{CO_{SD}}$ ,  $V_{CO_{95}}$ , and  $V_{CO_{83}}$  = average volumetric flow rate of the condenser outlet air during cooling mode testing for single-duct portable air conditioners; and at the 95 °F and 83 °F dry-bulb outdoor conditions for dual-duct portable air conditioners, respectively, in cubic feet per minute (cfm).

$V_{CI_{95}}$  and  $V_{CI_{83}}$  = average volumetric flow rate of the condenser inlet air during cooling mode testing at the 95 °F and 83 °F dry-bulb outdoor conditions for dual-duct portable air conditioners, respectively, in cfm.

$\rho_{CO_{SD}}$ ,  $\rho_{CO_{95}}$ , and  $\rho_{CO_{83}}$  = average density of the condenser outlet air during cooling mode testing for single-duct portable air conditioners, and at the 95 °F and 83 °F dry-bulb outdoor conditions for dual-duct portable air conditioners, respectively, in pounds mass per cubic foot (lb<sub>m</sub>/ft<sup>3</sup>).

$\rho_{CI_{95}}$  and  $\rho_{CI_{83}}$  = average density of the condenser inlet air during cooling mode testing at the 95 °F and 83 °F dry-bulb outdoor conditions for dual-duct portable air conditioners, respectively, in lb<sub>m</sub>/ft<sup>3</sup>.

$\omega_{CO_{SD}}$ ,  $\omega_{CO_{95}}$ , and  $\omega_{CO_{83}}$  = average humidity ratio of condenser outlet air during cooling mode testing for single-duct

portable air conditioners, and at the 95 °F and 83 °F dry-bulb outdoor conditions for dual-duct portable air conditioners, respectively, in pounds mass of water vapor per pounds mass of dry air (lb<sub>w</sub>/lb<sub>da</sub>).

$\omega_{CI_{95}}$  and  $\omega_{CI_{83}}$  = average humidity ratio of condenser inlet air during cooling mode testing at the 95 °F and 83 °F dry-bulb outdoor conditions for dual-duct portable air conditioners, respectively, in lb<sub>w</sub>/lb<sub>da</sub>.

For single-duct and dual-duct portable air conditioners, calculate the sensible component of infiltration air heat contribution according to:

$$Q_{s_{95}} = \dot{m} \times 60 \times [c_{p_{da}} \times (T_{ia_{95}} - T_{indoor}) + (c_{p_{wv}} \times (\omega_{ia_{95}} \times T_{ia_{95}} - \omega_{indoor} \times T_{indoor}))]$$

$$Q_{s_{83}} = \dot{m} \times 60 \times [(c_{p_{da}} \times T_{ia_{83}} - T_{indoor}) + (c_{p_{wv}} \times (\omega_{ia_{83}} \times T_{ia_{83}} - \omega_{indoor} \times T_{indoor}))]$$

Where:

$Q_{s_{95}}$  and  $Q_{s_{83}}$  = sensible heat added to the room by infiltration air, calculated at the 95 °F and 83 °F dry-bulb outdoor conditions in Table 1 of this appendix, in Btu/h.

$\dot{m}$  = dry air mass flow rate of infiltration air,  $\dot{m}_{SD}$  or  $\dot{m}_{95}$  when calculating  $Q_{s_{95}}$  and  $\dot{m}_{SD}$  or  $\dot{m}_{83}$  when calculating  $Q_{s_{83}}$ , in lb/m.

$c_{p_{da}}$  = specific heat of dry air, 0.24 Btu/lb<sub>m</sub> - °F.

$c_{p_{wv}}$  = specific heat of water vapor, 0.444 Btu/lb<sub>m</sub> - °F.

$T_{indoor}$  = indoor chamber dry-bulb temperature, 80 °F.

$T_{ia_{95}}$  and  $T_{ia_{83}}$  = infiltration air dry-bulb temperatures for the two test conditions in Table 1 of this appendix, 95 °F and 83 °F, respectively.

$\omega_{ia_{95}}$  and  $\omega_{ia_{83}}$  = humidity ratios of the 95 °F and 83 °F dry-bulb infiltration air, 0.0141 and 0.01086 lb<sub>w</sub>/lb<sub>da</sub>, respectively.

$\omega_{indoor}$  = humidity ratio of the indoor chamber air, 0.0112 lb<sub>w</sub>/lb<sub>da</sub>.

60 = conversion factor from minutes to hours.

Calculate the latent heat contribution of the infiltration air according to:

$$Q_{l_{95}} = \dot{m} \times 60 \times H_{fg} \times (\omega_{ia_{95}} - \omega_{indoor})$$

$$Q_{l_{83}} = \dot{m} \times 60 \times H_{fg} \times (\omega_{ia_{83}} - \omega_{indoor})$$

Where:

$Q_{l_{95}}$  and  $Q_{l_{83}}$  = latent heat added to the room by infiltration air, calculated at the 95 °F and 83 °F dry-bulb outdoor conditions in Table 1 of this appendix, in Btu/h.

$\dot{m}$  = mass flow rate of infiltration air,  $\dot{m}_{SD}$  or  $\dot{m}_{95}$  when calculating  $Q_{l_{95}}$  and  $\dot{m}_{SD}$  or  $\dot{m}_{83}$  when calculating  $Q_{l_{83}}$ , in lb/m.

$H_{fg}$  = latent heat of vaporization for water vapor, 1061 Btu/lb<sub>m</sub>.

$\omega_{ia_{95}}$  and  $\omega_{ia_{83}}$  = humidity ratios of the 95 °F and 83 °F dry-bulb infiltration air, 0.0141 and 0.01086 lb<sub>w</sub>/lb<sub>da</sub>, respectively.

$\omega_{indoor}$  = humidity ratio of the indoor chamber air, 0.0112 lb<sub>w</sub>/lb<sub>da</sub>, 60 = conversion factor from minutes to hours.

The total heat contribution of the infiltration air is the sum of the sensible and latent heat:

$$Q_{infiltration_{95}} = Q_{s_{95}} + Q_{l_{95}}$$

$$Q_{\text{infiltration}_83} = Q_{s_83} + Q_{l_83}$$

Where:

$Q_{\text{infiltration}_95}$  and  $Q_{\text{infiltration}_83}$  = total infiltration air heat in cooling mode, calculated at the 95 °F and 83 °F dry-bulb outdoor conditions in Table 1 of this appendix, in Btu/h.

$Q_{s_95}$  and  $Q_{s_83}$  = sensible heat added to the room by infiltration air, calculated at the 95 °F and 83 °F dry-bulb outdoor conditions in Table 1 of this appendix, in Btu/h.

$Q_{l_95}$  and  $Q_{l_83}$  = latent heat added to the room by infiltration air, calculated at the 95 °F and 83 °F dry-bulb outdoor conditions in Table 1 of this appendix, in Btu/h.

\* \* \* \* \*

[FR Doc. 2016-24869 Filed 10-13-16; 8:45 am]

BILLING CODE 6450-01-P

## FARM CREDIT ADMINISTRATION

### 12 CFR Chapter VI

#### Farm Credit Administration Board Policy Statements

**AGENCY:** Farm Credit Administration.

**ACTION:** Notice of policy statements and index.

**SUMMARY:** The Farm Credit Administration (FCA), as part of its annual public notification process, is publishing for notice an index of the 18 Board policy statements currently in existence. Most of the policy statements remain unchanged since our last **Federal Register** notice on November 2, 2015, except for one as discussed below on Equal Employment Opportunity and Diversity.

**DATES:** October 14, 2016.

**FOR FURTHER INFORMATION CONTACT:**

Dale L. Aultman, Secretary to Board, Farm Credit Administration, 1501 Farm Credit Drive, McLean, Virginia 22102-5090, (703) 883-4009, TTY (703) 883-4056; or

Mary Alice Donner, Senior Counsel, Office of General Counsel, Farm Credit Administration, 1501 Farm Credit Drive, McLean, Virginia 22102-5090, (703) 883-4020, TTY (703) 883-4020.

**SUPPLEMENTARY INFORMATION:** A list of the 18 FCA Board policy statements is set forth below. FCA Board policy statements may be viewed online at [www.fca.gov/handbook.nsf](http://www.fca.gov/handbook.nsf).

On August 8, 2016, the FCA Board updated FCA-PS-62 on, "Equal Employment Opportunity and Diversity." The policy was published in the **Federal Register** on August 12, 2016 (81 FR 53482). The policy was slightly edited at the Equal Employment

Opportunity Commission's recommendation to indicate that FCA begins prompt, thorough, and impartial investigations within 10 days of receiving notice of harassment allegations.

The FCA will continue to publish new or revised policy statements in their full text.

#### FCA Board Policy Statements

FCA-PS-34 Disclosure of the Issuance and Termination of Enforcement Documents

FCA-PS-37 Communications During Rulemaking

FCA-PS-41 Alternative Means of Dispute Resolution

FCA-PS-44 Travel

FCA-PS-53 Examination Philosophy

FCA-PS-59 Regulatory Philosophy

FCA-PS-62 Equal Employment Opportunity and Diversity

FCA-PS-64 Rules for the Transaction of Business of the Farm Credit Administration Board

FCA-PS-65 Release of Consolidated Reporting System Information

FCA-PS-67 Nondiscrimination on the Basis of Disability in Agency Programs and Activities

FCA-PS-68 FCS Building Association Management Operations Policies and Practices

FCA-PS-71 Disaster Relief Efforts by Farm Credit Institutions

FCA-PS-72 Financial Institution Rating System (FIRS)

FCA-PS-77 Borrower Privacy

FCA-PS-78 Official Names of Farm Credit Institutions

FCA-PS-79 Consideration and Referral of Supervisory Strategies and Enforcement Actions

FCA-PS-80 Cooperative Operating Philosophy—Serving the Members of Farm Credit System Institutions

FCA-PS-81 Ethics, Independence, Arm's-Length Role, Ex Parte Communications and Open Government

Dated: October 6, 2016.

**Dale L. Aultman,**

Secretary, Farm Credit Administration Board.

[FR Doc. 2016-24680 Filed 10-13-16; 8:45 am]

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## DEPARTMENT OF TRANSPORTATION

### Federal Aviation Administration

#### 14 CFR Part 39

[Docket No. FAA-2016-5872; Directorate Identifier 2016-NE-11-AD; Amendment 39-18681; AD 2016-20-15]

RIN 2120-AA64

#### Airworthiness Directives; General Electric Company Turbofan Engines

**AGENCY:** Federal Aviation Administration (FAA), DOT.

**ACTION:** Final rule.

**SUMMARY:** We are adopting a new airworthiness directive (AD) for all General Electric Company (GE) GENx-1B64/P2, -1B67/P2, -1B70/P2, -1B70C/P2, -1B70/75/P2, and -1B74/75/P2 turbofan engines with engine assembly, part number (P/N) 2447M10G01 or P/N 2447M10G02, installed. This AD was prompted by a report of a significant fan rub event. This AD requires rework of the engine fan stator module assembly. We are issuing this AD to prevent failure of the fan blades and the load reduction device, loss of power to one or more engines, loss of thrust control, and loss of the airplane.

**DATES:** This AD is effective November 18, 2016.

The Director of the Federal Register approved the incorporation by reference of certain publications listed in this AD as of November 18, 2016.

**ADDRESSES:** For service information identified in this final rule, contact General Electric Company, GE Aviation, Room 285, 1 Neumann Way, Cincinnati, OH 45215; phone: 513-552-3272; email: [aviation.fleetsupport@ge.com](mailto:aviation.fleetsupport@ge.com). You may view this referenced service information at the FAA, Engine & Propeller Directorate, 1200 District Avenue, Burlington, MA 01803. For information on the availability of this material at the FAA, call 781-238-7125. It is also available on the internet at <http://www.regulations.gov> by searching for and locating Docket No. FAA-2016-5872.

#### Examining the AD Docket

You may examine the AD docket on the Internet at <http://www.regulations.gov> by searching for and locating Docket No. FAA-2016-5872; or in person at the Docket Management Facility between 9 a.m. and 5 p.m., Monday through Friday, except Federal holidays. The AD docket contains this AD, the regulatory evaluation, any comments received, and other information. The address for the